

Tractable Predictive Control Strategies for Heating Systems in Buildings

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Summary

- 1 Introduction
- 2 Model Based Predictive Control
- 3 Case study
- 4 Simulation Results
- 5 Conclusion and outlook

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- 1 Introduction
- 2 Model Based Predictive Control
 - Energy Context
 - “Sustainable Building and Innovation” Project
- 3 Case study
- 4 Simulation Results
- 5 Conclusion and outlook

Energy context

- Non-residential building sector is one of the most **energy-consuming sectors**
- Needs to reduce consumption have led to new low-consumption non-residential buildings
 - ⇒ Then heating and cooling can be significantly reduced
- But:
 - Low consumption buildings have **limited heat power** and **high inertia**
 - Non-residential buildings have known **intermittent occupation**
- ⇒ Therefore important **thermal discomfort** and **energy waste** can appear:
 - Thermal comfort takes too long to be good after a none-occupancy period
 - Internal heat gain can cause energy waste and overheating
 - Energy consumption until the end of occupancy is not always required

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“Sustainable Building and Innovation” Project

- Financial support and partnerships
 - Research chair on “Sustainable Building and Innovation”
 - Financial support of Bouygues Construction
 - Partnership between research laboratories of Supélec (French graduate school of engineering) and the CSTB (French Scientific and Technical Center for Building)



- Objectives
 - Develop a simulation model of a low-energy non-residential building using Simbad
 - Evaluate the impact of intermittent occupation on both the thermal comfort of the occupants and energy consumption
 - Propose predictive control algorithms to manage heating efficiently, taking into account intermittency
 - Propose solutions which are computationally tractable in order to be implanted in industrial embedded system

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- 2 **Model Based Predictive Control**
 - Control structure
 - Predictive controller MPC 1
 - Predictive controller MPC 2
- 3 Case study
- 4 Simulation Results
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PI-control structure for the heating system

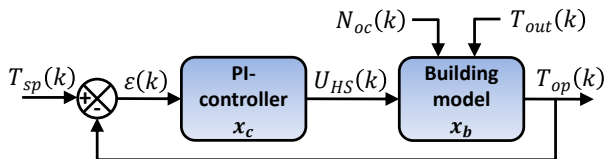
- Linear thermal model of the building

$$\begin{cases} \mathbf{x}_b(k+1) = \mathbf{A}_b \mathbf{x}_b(k) + \mathbf{B}_b \begin{bmatrix} U_{HS}(k) \\ T_{out}(k) \\ N_{oc}(k) \end{bmatrix} \\ T_{op}(k) = \mathbf{C}_b \mathbf{x}_b(k) \end{cases} \quad (1)$$

- PI-control of the heating system

$$\begin{cases} \mathbf{x}_c(k+1) = \mathbf{A}_c \mathbf{x}_c(k) + \mathbf{B}_c \varepsilon(k) \\ U_{HS}(k) = \mathbf{C}_c \mathbf{x}_c(k) + \mathbf{D}_c \varepsilon(k) \end{cases} \quad (2)$$

- Closed loop of the PI-control



Physical variables

T_{sp} → set-point temperature

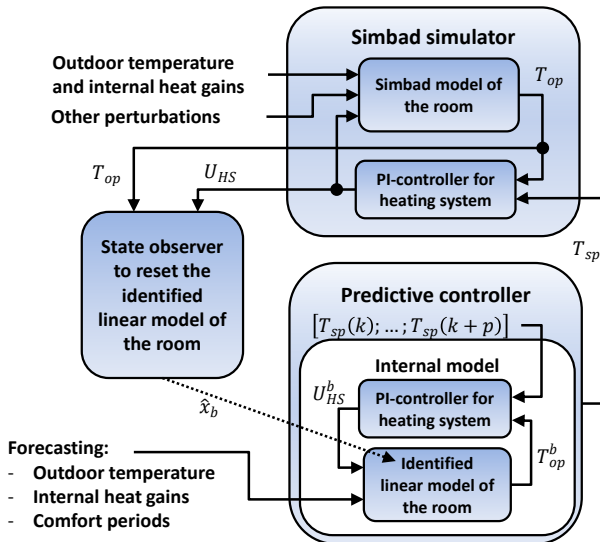
T_{op} → operative temperature

T_{out} → outdoor temperature

U_{HS} → heat power

N_{oc} → number of occupants

MPC-control structure for the heating system



1st predictive controller (with linear optimization)

- On-line linear optimization problem

Problem

At a time k , given $\mathbf{x}_b(k)$, $\mathbf{x}_c(k)$, the optimization problem is:

$$\min_{\mathbf{T}_{SP}(k:k+N_h-1)} \sum_{j=0}^{N_h-1} U_{HS}(k+j), \quad (3)$$

s.t. $\forall j = 1..N_h \setminus N_{oc}(k+j) \neq 0$

$$T_{op,min} \leq T_{op}(k+j) \leq T_{op,max} \quad (4)$$

$$0 \leq U_{HS}(k+j) \leq U_{HS}^{max} \quad (5)$$

T_{op} and U_{HS} are computed according to linear models of the room and PI-controller with the prediction of:

- $\mathbf{T}_{SP}(k : k + N_h - 1)$ (output of the MPC controller),
- $\mathbf{T}_{out}(k : k + N_h - 1)$ (flat prediction)
- $\mathbf{N}_{oc}(k : k + N_h - 1)$ (supposed known)

2nd predictive controller (without on-line optimization)

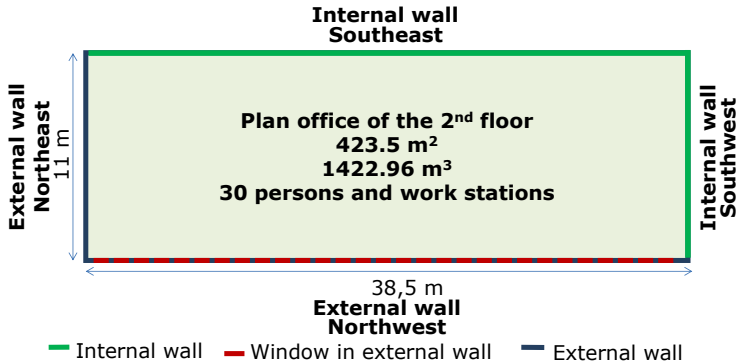
- Logical decision based on the prediction model
- No optimization toolbox needed
- The controller answers two questions:
 - Case 1: During inoccupancy, when switching the heating system on to ensure the thermal comfort for the next working hours, now or later ? (later if possible)
 - Case 2: During occupancy, when switching the heating system off to ensure the thermal comfort until the last occupancy hour, now or later ? (now if possible)
- Algorithms to solve these problems:
 - Case 1 : One simulation with input: $\mathbf{T}_{SP}(k : k + N_h) = [0, T_{SP}, T_{SP}, \dots, T_{SP}]^T$
 \Rightarrow if $T_{op}(k : k + N_h) \geq T_{op,min}(k + N_h)$ then apply: $T_{SP}(k) = 0$
 else apply: $T_{SP}(k) = T_{SP}$
 - Case 2 : One simulation with input: $\mathbf{T}_{SP}(k : k + N_h) = [0, T_{SP}, T_{SP}, \dots, T_{SP}]^T$
 \Rightarrow if $\forall i = 1, \dots, N_h, T_{op}(k + i) \geq T_{op,min}(k + i)$ then apply: $T_{SP}(k) = 0$
 else apply: $T_{SP}(k) = T_{SP}$

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- 3 **Case study**
 - Geometry, thermal parameters and systems
 - Scenario Inputs
- 4 Simulation Results
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Geometry of the building and window parameters

- One room in a low-consumption energy building used for office work
- Second floor of the 4-floor building
- Two external walls and two internal walls
- 28 windows on the Northwest side



Thermal parameters for walls and windows

THERMAL PARAMETERS FOR WALLS					
Wall	Layer	Thickness m	Density $\text{kg}\cdot\text{m}^{-3}$	Capacity $\text{J}\cdot\text{kg}^{-1}\cdot\text{K}^{-1}$	Conductivity $\text{W}\cdot\text{m}^{-1}\cdot\text{K}^{-1}$
External	Reinforced concrete	0.2	2150	1008	1.650
	Rockwool "Rockfaçade [®] "	0.12	39	1030	0.036
	Unventilated air gap	0.02	1	1000	0.130
	Ventilated air gap	0.022	1	1000	0.192
	Terra cotta "Terreal Zéphir [®] "	0.014	2286	1008	0.98
Internal	Drywall "BA13"	0.0125	825	1008	0.25
	Unventilated air gap	0.025	1	1000	0.155
	Drywall "BA13"	0.0125	825	1008	0.25
Floor	Concrete	0.31	2350	880	2.3

THERMAL PARAMETERS FOR WINDOWS	
Parameters	Value
Thermal diffusivity ($\text{W}\cdot\text{m}^{-2}\cdot\text{K}^{-1}$)	1.8
Solar absorption g factor	0.095
Light transmission	0.42
Emissivity of exterior side	0.71
Emissivity of interior side	0.095

Systems for lighting and HVAC

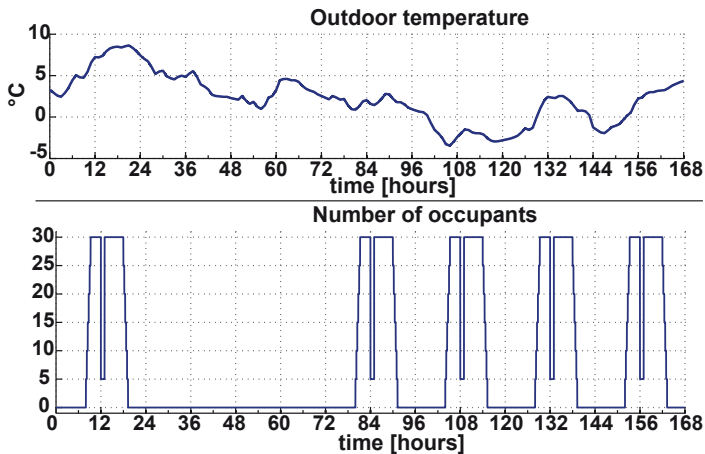
- Lighting system

PARAMETERS OF THE LIGHTING SYSTEM		
Name	Unit	Value
Total lighting power	W	40
Illuminance efficiency	lm/W	88
Luminaire mean efficiency	–	0.8
Luminaire maintenance factor	–	1.11
Lighting heat gain	–	0.25

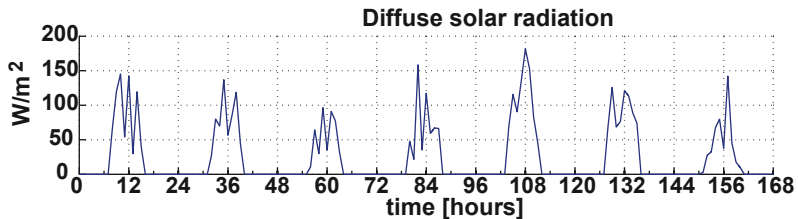
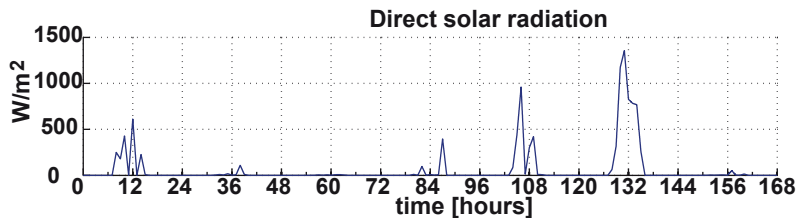
- HVAC system

- Mechanical ventilation with heat recovery ($\varepsilon = 84\%$)
- Air flow during occupancy : $0.3454 \text{ kg}\cdot\text{s}^{-1}$
- Air flow during inoccupancy : $0.03454 \text{ kg}\cdot\text{s}^{-1}$
- Heat power : 0 to 12.588 kW ($8.8 \text{ W}\cdot\text{m}^3$)

Outdoor temperature and number of occupants



Solar radiation



Summary

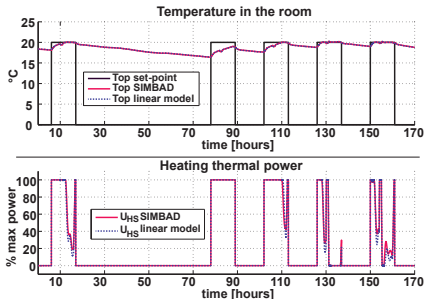
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 - Identification of a linear model
 - Evaluation of thermal comfort
 - Simulation results of PI-controller strategies
 - Simulation results of Predictive Control MPC 1
 - Simulation results of Predictive Control MPC 2
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Identification of a linear model

- Identification of a 4-order linear model of the room (open loop), using the numerical algorithm for subspace state-space systems, based on dataset generated with Simbad
- Validation of the closed-loop model (with the PI-controller) considering 2 scenarios
 - V1: heating control always switched on
 - V2: heating control works during occupancy periods

VALIDATION OF THE CLOSED-LOOP MODEL

Criterion	Unit	V1	V2
<i>FIT</i> of T_{op}	%	95.6	96.7
<i>FIT</i> of U_{HS}	%	80.1	95.8
Total energy difference	%	-0.62	0.26



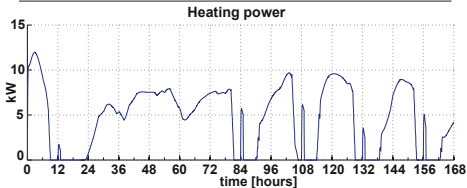
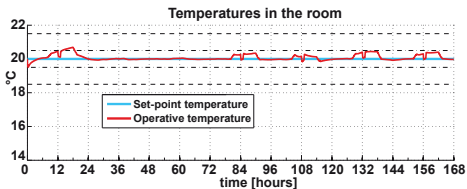
Evaluation of thermal comfort

- Thermal comfort is evaluated according to % of time in three comfort domains for the operative temperature T_{op}
 - D_1 : the optimal comfort domain (a 1°C width temperature band, centered around the set-point).
 - D_2 : the low discomfort domain.
 - D_3 : the high discomfort domain, when the occupants feel an important thermal discomfort.

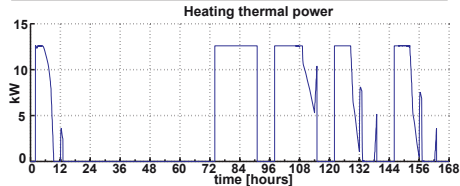
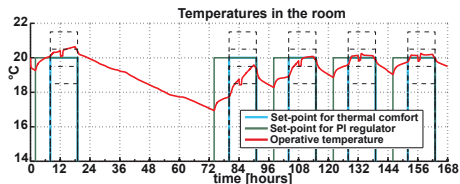
THERMAL COMFORT DOMAINS	
Name	Conditions
D_1	$\{T_{sp} + 0.5 > T_{op} > T_{sp} - 0.5\}$
D_2	$\{T_{sp} + 1.5 > T_{op} > T_{sp} + 0.5\}$ $\cup \{T_{sp} - 0.5 > T_{op} > T_{sp} - 1.5\}$
D_3	$\{T_{op} > T_{sp} + 1.5\}$ $\cup \{T_{op} < T_{sp} - 1.5\}$

Results of PI-controller strategies

- PI-S1: Heating switched on 24/7
- Energy consumed: 797 kWh
- % of time with an optimal comfort: 91.1%

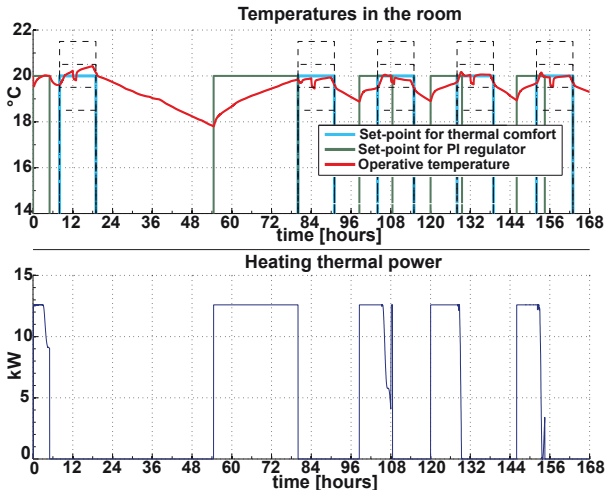


- PI-S2: 6-hour anticipation
- Energy consumed: 700 kWh
- % of time with an optimal comfort: 70.33%



Results of the 2nd Predictive Controller (MPC 2)

- Energy consumed: 700 kWh (-12.2% than PI-S1, equal than PI-S2)
- % of time with an optimal comfort: 99.3% (+9% than PI-S1, +41.2% than PI-S2)



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Conclusion and outlook

● Conclusions

- Development of a simple but intelligent predictive strategy
- Preservation of local controllers
- Computationally tractable algorithms
 - on-line linear optimization (MPC 1)
 - no-online optimization (MPC 2)
- Significant reduction of energy use and better comfort level

● Future works

- Estimation of operative temperature without any radiant measurement
- Use of PMV instead of operative temperature
- Generalization for multi-zone buildings
- Control of multiple systems (blinding system. . .)
- Integration of variable cost of energy
- Experimental validation in a real building

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