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# Sensorless Force Feedback Joystick Control for Teleoperation of Construction Equipment

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*This paper aims to develop an innovative approach named sensorless force feedback joystick control for teleoperation of construction equipment. First, a force sensorless supervisory controller is designed with two advanced modules: a neural network-based environment classifier to estimate environment characteristics without requiring a force sensor and, a fuzzy-based force feedback tuner to generate properly a force reflection to the joystick. Second, two local robust adaptive controllers are simply built using neural network and Lyapunov stability condition to ensure desired task performances at both master and slave sites. A teleoperation system is setup to demonstrate the applicability of the proposed approach.*

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## NOMENCLATURE

$F_h$  = operator command  
 $F_{dr}$  = desired reflected force  
 $P_{dr}$  = desired reflected pressure  
 $X_m$  = joystick shaft rotation  
 $X_{ds}$  = slave desired movement  
 $\lambda_p$  = conversion ratio between joystick and slave commands  
 $u_s$  = slave controller output  
 $X_s$  = slave actuation  
 $F_e$  = loading force from environment  
 $k_e$  = environment stiffness  
 $c_e$  = environment damping coefficient  
 $g_k$  = scaling factor of environment stiffness  
 $g_c$  = scaling factor of environment damping coefficient  
 $k_f$  = force feedback gain  
 $\lambda_f$  = conversion ratio between  $F_{dr}$  and  $P_{dr}$   
 $u_m$  = master controller output  
 $y^{desired}$  = master/slave desired response  
 $y^{actual}$  = master/slave response  
 $e$  = local controller control error

## 1. Introduction

Nowadays, teleoperation takes a key role in remote manipulation that allows users ability to perform naturally manual tasks at environments away from the normal human reach, such as undersea applications, hazardous assignments, minimally invasive surgical systems. In common, control schemes for teleoperation systems can be classified as either compliance control or bilateral control. In the compliance control,<sup>1-4</sup> the contact force sensed by the slave device is not reflected back to the operator, but is used for the compliance control of the slave device. On the contrary, in the bilateral control,<sup>5-12</sup> the contact force is reflected back to the operator. The operator is able to achieve physical perception of interactions at the remote site similar as directly working at this site. Consequently, it improves the accuracy and safety in the tele-operated manipulation. In addition, the force reflection can enhance the human operator's task performance, for example in terms of task completion time, total contact time. Thus, the bilateral control has drawn a lot of attention.<sup>13,14</sup>

Two common architectures for a bilateral teleoperation system are known as: position-position and force-position architectures.<sup>7,13</sup> In the first architecture, the master position is passed to the slave and the slave position is passed again to the master. Then, the reflected force applied to the operator is derived from the position difference between the two



Furthermore, with the pneumatic solution, the FFM is able to work in safe conditions without damages from the operator.

The proposed algorithm is composed of the three main routines: force sensorless supervisory controller and two local robust adaptive controllers. During the system operation, the human operator applies a force,  $F_h$ , to the joystick handle to provide a command for the slave. By this way, the joystick shaft rotation,  $X_m$ , is detected and converted into the slave command,  $X_{ds}$ , via a suitable ratio,  $\lambda_p$ . The SRAC attempts to make the slave execute the given command with high accuracy regardless any impact (unknown loading force) from the environment,  $F_e$ . Next, the command ( $u_s$ ) and actual response ( $X_s$ ) of the slave are acquired and input to the FSSC. This supervisory controller takes part in classifying the environment characteristics to estimate the loading condition at the slave site ( $F_e$ ). Consequently, based on this estimated load value, the desired reflected force,  $F_{dr}$ , is properly produced. This resultant is converted approximately to a desired reflected pressure,  $P_{dr}$ , for the FFM using a transformed factor,  $\lambda_f$ . Finally, the MRAC drives the FFM (with command  $u_m$ ) to create the desired pressure and, successively, creates the desired reflected force on the operator hand via the joystick handle. By this way, the operator can attain the truthful perception of the loading condition at the slave manipulator.

## 2.2 Teleoperation test rig

In order to evaluate the effectiveness of the SFFJC approach, a simple 1-DOF teleoperation test rig has been designed as Fig. 2. From this figure, the system consists of a master box, a slave – environment box in which a slave manipulator is connected to an environment simulator, and the SFFJC built in a personal computer (PC). The experimental system has been then fabricated as displayed in Fig. 3. In this study, only wired communication method is considered to develop the proposed control approach. To perform the wired communication between the SFFJC and the master/slave devices, a National Instrument (NI) multifunction data acquisition (DAQ) device with a suitable number of digital and analog channels is chosen.

As shown in Fig. 3(a), although two joysticks are installed in the master box, only one is used to develop the SFFJC while the other one will be used for the future research. The selected joystick is integrated with a FFM as described in Fig. 3(b). Rotation of the joystick handle generated by the human interaction is detected by a potentiometer attached at the pivot shaft of the joystick. This action is then converted into electrical commands to send to the SFFJC to drive the slave. For the force feedback concept as presented in the previous section, the FFM constructed by a mini pneumatic rotary actuator and two bias springs is attached to the opposite side of the joystick handle. Due to the limited free space of the joystick rotating mechanism, a slider-crank mechanism needs to be used to link the joystick rotating mechanism with the pneumatic rotary actuator. A proportional flow control valve (control valve 1) is used to drive the rotary actuator. And a pressure sensor is attached to the rotary actuator to perform the closed control loop and subsequently, to regulate any desired reflecting forces.

The slave – environment box setup is shown in Fig. 3(c) and Fig. 3(d). The slave employs an asymmetrically pneumatic cylinder as its manipulator. To perform the closed loop control, the cylinder is driven by another proportional control valve (control valve 2) and the piston displacement is measured by a linear variable displacement transducer

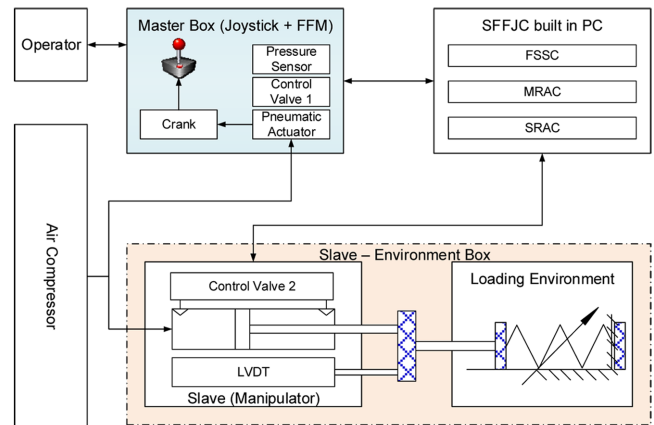


Fig. 2 Design layout for the teleoperation test rig

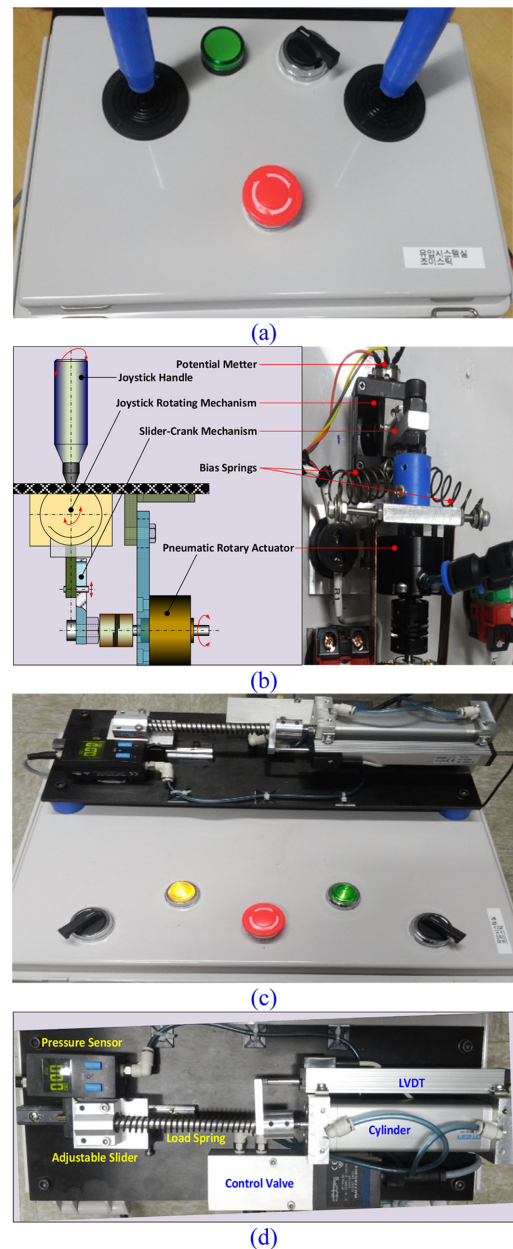


Fig. 3 Teleoperation test rig: (a) Master box, (b) FFM Design, (c) Slave - environment box, (d) Slave - environment connection

Table 1 Specifications of the system components

Parts	Type	Characteristics
Rotary actuator	CRB1BW15 90-D	Max. torque: 0.9 Nm
Pressure sensors	SDE1-D10-G2-W18	Pressure range: 0-10 bar Stroke: 100 mm
Pneumatic cylinder	CDC-20	Bore diameter: 20 mm Rod diameter: 8 mm
Proportional control valves	MPYE-5-1/4-010B	Control voltage range: 0-10 VDC
LVDT	Novotechnik TR100	Measurement range: 0-100 mm
Springs	Types: 1; 2; 3; 4; 5	Stiffness N/m: 500; 1000; 1500; 2000; 2500

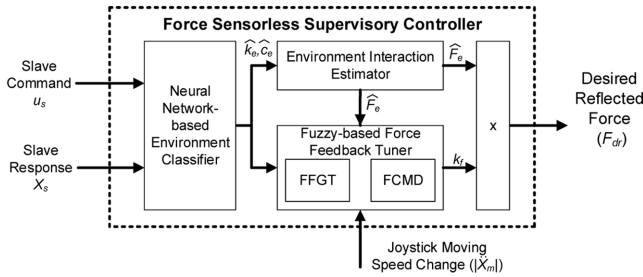


Fig. 4 Configuration of supervisory controller

(LVDT). Moreover, second pressure sensor is attached to support the FSSC design. The environment simulator is presented by a spring-slider mechanism in which the spring stiffness and initial load can be manually adjusted (by changing the spring and the fixed position of the slider) to simulate different load conditions. An air compressor is installed as the main power source to supply the pressurized air for both the FFM and the slave cylinder. According to the design, the main components are properly chosen as listed in Table 1.

### 3. Force Sensorless Supervisory Controller

#### 3.1 Supervisory control architecture

Configuration of the supervisory controller is described in Fig. 4.

The FSSC consists of two main modules, NNEC and FFFT, with three inputs and one output. The first two inputs, the slave command ( $u_s$ ) and its response ( $X_s$ ), are received by the NNEC. Here, the classifier is built as a learning vector quantitative neural network (LVQNN) capable of detecting the working environment. Without loss of generality, the interactive environment can be represented by two factors: damping  $c_e$  and stiffness  $k_e$ . Thus, there are two outputs from the NNEC which are the predicted values of the environment damping and stiffness,  $\hat{c}_e$  and  $\hat{k}_e$ , respectively. These outputs are then fed into the environment interaction estimator to estimate the contact force between the manipulator and environment ( $\hat{F}_e = k_e X_s + c_e \dot{X}_s$ ). These outputs combined with another signal - joystick moving speed change ( $\dot{X}_m$ ) are input to the FFFT module.

The FFFT module includes a fuzzy feedback gain tuner (FFGT) to tune the force feedback gain ( $k_f$ ) based on the estimated damping and stiffness. Additionally, the impacts of the estimated contact force and

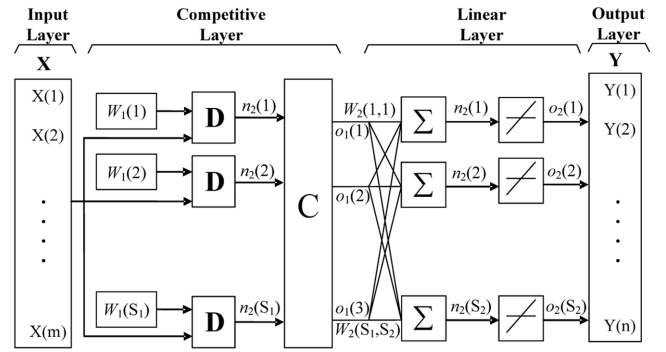


Fig. 5 General structure of a LVQNN

the human-joystick dynamics represented by the joystick moving speed change are taken into account for online refining the gain  $k_f$  to ensure the stability performance of the mechanism. To fulfill this requirement, a fuzzy cognitive map-based decision (FCMD) tool is employed. The final output value of  $k_f$  is used to produce the desired force feedback (or desired reflect force,  $F_{dr}$ ) which needs to be applied to the joystick.

#### 3.2 Neural network-based environment classifier

##### 3.2.1 Learning vector quantitative neural network

There exist many classification techniques successfully developed to support machine learning. Well-known techniques can be listed as logic-based algorithm as decision tree, support vector machines, static learning mechanism as Bayesian network, instance-based learning scheme as learning vector quantization (LVQ) or self-organizing map (SOM), and artificial intelligence-based method (fuzzy, genetic or neural network).<sup>16,17</sup> Comparing to the others, intelligence-based methods offer higher adaptability and better performance in dealing with complex and partial unknown/unknown systems with limited and noisy data.<sup>16,17</sup> Among intelligent solutions, neural network (NN) is realized as the powerful tool that provides higher flexibility and stronger capability than fuzzy logic while requires less computational effort than generic algorithm.<sup>17,18</sup> Generally, neural network can be classified accordingly to the learning process: supervised and unsupervised learning. Supervised learning is training using desired responses for given stimuli while unsupervised learning is classification by “clustering of stimuli, without specified responses. However, these methods normally require a heavy training process. An LVQNN as depicted in Fig. 5 therefore is considered as feasible classification tool.

The LVQNN is a hybrid network which uses the advanced behaviors of both competitive learning networks and bases on the LVQ and Kohonen SOM to form the classification with the high speed. An LVQNN generally contains four layers: input layer with  $m$  nodes, first hidden layer named competitive layer with  $S_1$  nodes, second hidden layer named linear layer with  $S_2$  nodes, and output layer with  $n$  nodes (in this case,  $S_2 \equiv n$ ).

The larger the hidden layer, the more clusters the competitive layer can learn, and the more complex mapping of input to target classes can be made<sup>19</sup>. With proper selection of the structure and training of the weighting factors, the LVQNN can classify any system information.

Each node,  $n_j$ , in the competitive layer is computed using the so-called nearest-neighbor method in which the Euclidean distance weight



function,  $D$ , is employed:

$$n_j = D(X, W_1(j)) = \frac{1}{\sqrt{\sum_{i=1}^m (X(i) - W_1(j, i))^2}}, \quad j = 1, \dots, S_1 \quad (1)$$

where  $X$  is the input vector;  $W_1(j, i)$  is the weight of node  $j^{\text{th}}$  in the competitive layer corresponding to element  $i^{\text{th}}$  of the input vector.

Next, the derived Euclidean distances are fed into function  $C$  which is a competitive transfer function. This function returns an output vector  $o_1$ , with 1, where the net input vector reaches its maximum value, and 0 elsewhere. The achieved output vector is then input to the linear layer to produce a vector  $o_2$ , where each element can be computed as

$$\begin{aligned} Y(k) &= o_2(k) = k_W(k)n_2(k) \\ &= k_W(k) \sum_{j=1}^{S_1} W_2(k, j) o_1(j), \quad k = 1, \dots, n, \quad (n \equiv S_2) \end{aligned} \quad (2)$$

where  $W_2(k, j)$  is the weight of node  $k^{\text{th}}$  in the linear layer corresponding to element  $j^{\text{th}}$  of the competitive output vector;  $k_W(k)$  is the linearized gain of node  $k^{\text{th}}$  in the linear layer.

In the learning process, the weights of LVQNN are updated by the well-known Kohonen rule as

$$\begin{cases} \text{IF: } X \text{ is classified correctly:} \\ W_1^{t+1}(j) = W_1^t(j) + \mu(X - W_1^t(j)) \\ \text{Else: (X is classified incorrectly)} \\ W_1^{t+1}(j) = W_1^t(j) - \mu(X - W_1^t(j)) \end{cases}, \quad j = 1, \dots, S_1 \quad (3)$$

where  $\mu$  is the positive learning ratio and is decreased with respect to the number of training iterations ( $n_{\text{iteration}}$ ),  $\mu = n_{\text{iteration}}$ .

### 3.2.2 LVQNN design for NNEC

A LVQNN is employed to construct the NNEC in order to detect the environment characteristics in an online manner. For a LVQNN design, it is important to determine the input vector size and how many sequences of data use. Here, with the limited number of input information, the NNEC is built with: an input vector consisting of current and historical values of four signals: the slave driving command,  $\{u_s^{(0)}, u_s^{(-1)}, \dots, u_s^{(-g)}\}$ , manipulator response,  $\{X_s^{(0)}, X_s^{(-1)}, \dots, X_s^{(-p)}\}$ , and their derivatives,  $\{du_s^{(0)}, du_s^{(-1)}, \dots, du_s^{(-m)}\}$  and  $\{dX_s^{(0)}, dX_s^{(-1)}, \dots, dX_s^{(-q)}\}$ , respectively; an output vector containing the estimated values of the damping and stiffness,  $\hat{c}_e$  and  $\hat{k}_e$ , representing the environment class  $e^{\text{th}}$  ( $class_e$ ). Configuration of the proposed classifier is described in Fig. 6. Due to uncertainties of the working environment, it is necessary to derive an algorithm to smoothly shift between different environment classes. Thus, a so-called smooth switching algorithm is proposed to determine the current environment based on the current class output from the LVQNN and the last detected class as

$$\begin{cases} \hat{c}_e(t) = \lambda \times \hat{c}_e|_{class_e(t-1)} + (1-\lambda) \times \hat{c}_e|_{LVQNN(t)} \\ \hat{k}_e(t) = \lambda \times \hat{k}_e|_{class_e(t-1)} + (1-\lambda) \times \hat{k}_e|_{LVQNN(t)} \end{cases} \quad (4)$$

where  $\lambda$  is called forgetting factor.

Additionally, to avoid influences of noises on the NNEC performance, the forgetting factor is online tuned according to the changing speed of the classifier outputs,  $v_Y$ , which is defined by the number of sampling periods when the LVQNN outputs change continuously. The procedure to tune this factor can be expressed as:

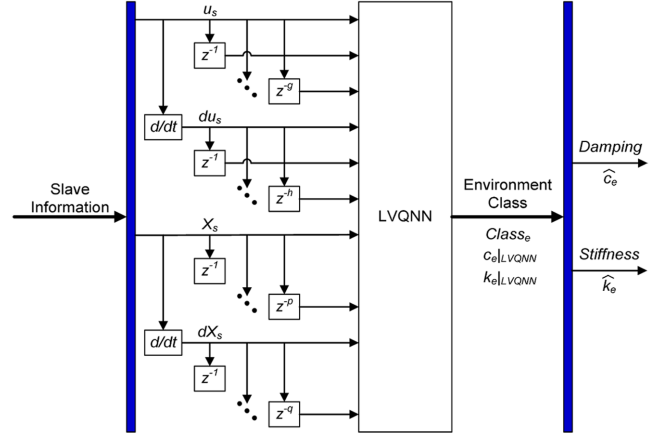


Fig. 6 Configuration of the NNEC

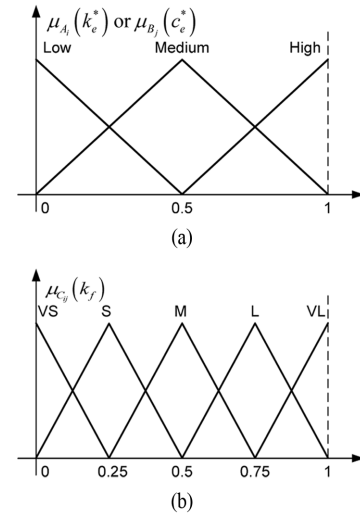


Fig. 7 FFGT MFs Design: (a) Input variables, (b) Output variables

**Step 1:** set initial value for the forgetting factor,  $\lambda = 0.5$ ; define a small positive thresholds,  $0 < \gamma_1 < \gamma_2$ , for  $v_Y$ .

**Step 2:** for each working step, check  $v_Y$  and update  $\lambda$  by comparing  $v_Y$  with its thresholds using the following rule:

+ If: ( $v_Y < \gamma_2$ ), Then:  $\lambda(t+1) = \lambda(t+1)/2$  and reset  $v_Y = 0$ ;

+ Else If: ( $v_Y \geq \gamma_1$ ) & ( $v_Y(t) \leq \gamma_2$ ), Then:  $\lambda(t+1) = \lambda(t+1) \times 2$  and reset  $v_Y = 0$ ;

+ Otherwise,  $\lambda(t+1) = \lambda(t)$ .

### 3.3 Fuzzy-based force feedback tuner

Although the environment characteristics are estimated using the NNEC, it is difficult to determine properly the FFG which is normally based on experience or prior knowledge about teleoperation systems. In decision making, many research works have shown that fuzzy logic which can take place of a skilled human operator is a feasible tool.<sup>20</sup> Thus, in this research, the fuzzy-based feedback gain tuner and fuzzy cognitive map-based decision tool are proposed to compute the FFG.

#### 3.3.1 Fuzzy feedback gain tuner

The FFGT is designed with two inputs, denoted as  $k_e^*$ ,  $c_e^*$ , and single

Table 2 Rule table design for the FFGT

FFG ( $k_f$ )	Normalized estimated stiffness $k_e^*$			
	Low	Medium	High	
Normalized estimate	Low	VL	L	M
damping $c_e^*$	Medium	L	M	S
	High	M	S	VS

output,  $k_f$ . The inputs are normalized values of the NNEC outputs:

$$k_e^* = g_k \times \hat{k}_e, c_e^* = g_c \times \hat{c}_e, k_e^* \in [0,1], c_e^* \in [0,1] \quad (5)$$

where:  $g_k$  and  $g_c$  are the scaling factors.

The FFGT inspects the incoming system states, and transforms them into linguistic variables. Here, the linguistic variables of the stiffness and damping inputs are described in terms of ‘Low’, ‘Medium’, and ‘High’. Meanwhile, the gain output is described as ‘VS’ (Very Small), ‘S’ (Small), ‘M’ (Medium), ‘L’ (Large), ‘VL’ (Very Large). Triangle membership functions (MFs) are selected to represent these variables. These MFs are distributed over the FFGT input/output ranges as shown in Fig. 7. Next, the rule base is utilized to interpret expert knowledge in a useful way. It contains a set of conditional sentences in the form of (6). Subsequently, the rule table for this FGT is established in Table 2.

$$\text{Rule } R_{ij} : \text{if } k_e^* \text{ is } A_i \text{ and } c_e^* \text{ is } B_j \text{ then } k_f \text{ is } C_{ij}, (i, j = 1, 2, 3) \quad (6)$$

where  $A_i$ ,  $B_j$  and  $C_{ij}$  are the fuzzy subsets of the variables  $k_e^*$ ,  $c_e^*$  and  $k_f$ .

The fuzzy inference is performed using the MAX-MIN operator. Let  $\mu_{R_{ij}}(k_f)$  be MF of a subset of the output which is the result of rule  $R_{ij}$ . Then, it can be obtained by

$$\mu_{R_{ij}}(k_f) = \min(\mu_{A_i}(k_e^*), \mu_{B_j}(c_e^*), \mu_{C_{ij}}(k_f)) \quad (7)$$

Successively, the results of the nine rules are compared together to infer the final output MF  $\mu_R(k_f)$  using the MAX operator:

$$\mu_R(k_f) = \max(\mu_{R_{11}}(k_f), \mu_{R_{12}}(k_f), \dots, \mu_{R_{33}}(k_f)) \quad (8)$$

The fuzzy decoder is finally used to produce the FFG gain,  $k_f$ :

$$k_f = \text{Defuzzify}(\mu_R(k_f)) \times \left( \frac{\bar{k}_f - \underline{k}_f}{2} \right) + \underline{k}_f \quad (9)$$

where  $\bar{k}_f$  and  $\underline{k}_f$  are in turn the maximum and minimum values of the FFG;  $\text{Defuzzify}(\bullet)$  is the defuzzifier function which performs defuzzification by using the center average method.

### 3.3.2 Fuzzy cognitive map-based decision tool

Fuzzy cognitive maps (FCMs) are able to deal with processes like decision making that is based on human reasoning process<sup>21</sup> and therefore, have been successfully used for many applications, ranging from medical fields, agricultural applications and environmental areas to energy problems.<sup>21-23</sup> In this research, a FCM-based decision tool is developed to refine the FFG when considering the impacts of the estimated contact force,  $F_e$ , joystick moving speed change,  $|\dot{x}_m|$ , and upper limit of the gain  $k_f$ . This decision tool is built with a number of FFG decisive factors and knowledge of experts, along with the proper

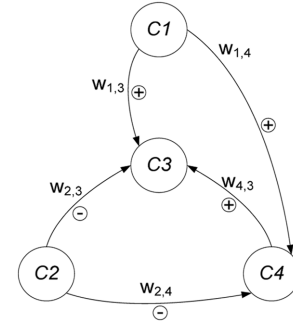


Fig. 8 Fuzzy cognitive map design for FFG decisive factors

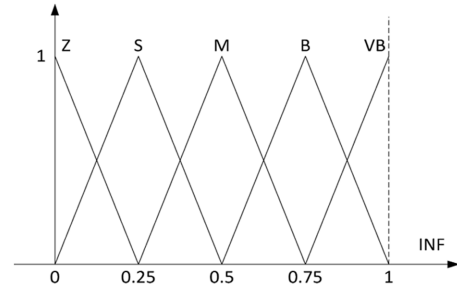


Fig. 9 MF design for the node influence

selection of fuzzification and defuzzification functions.

### 3.3.3 Fuzzy cognitive map

An FCM designed for a system is graphically represented by a frame of nodes and connection edges. The nodes (or concepts) stand for different behaviours of the system. Each node can be an input/output variable, a state, or an event of the system. In addition, these nodes also interact with each other and therefore, are capable of representing the system dynamics. The interactions between nodes are modelled by connection edges of which the directions and weights indicate the directions and degrees of the causal relationships, respectively.

Each node can be denoted as  $C_i$  with  $i = 1, \dots, N$  ( $N$  is the number of FCM nodes) and characterized by a specific value,  $x_i$ , which is fuzzified from the real system behaviour value into the closed universe  $[0, 1]$ . Between each two nodes,  $i^{\text{th}}$  and  $j^{\text{th}}$ , there are three possible relationships which are known as positive, negative or neutral causality and can be expressed by weight factors,  $w_{ij}$ , which are interpreted using linguistic variables in a normalized range  $[-1, 1]$ . A weight set of a generic FCM therefore can be defined as

$$W = \begin{bmatrix} w_{1,1} & \cdots & w_{1,N} \\ \vdots & \ddots & \vdots \\ w_{N,1} & \cdots & w_{N,N} \end{bmatrix}; w_{ij} \in [-1, 1]; i, j = 1, \dots, N \quad (10)$$

The value,  $x_i$ , of node  $C_i$  at step  $(k+1)^{\text{th}}$  can be computed based on the influence of the other interconnected nodes,  $C_j$ , on node  $C_i$ , as

$$x_i^{k+1} = f \left( x_i^k + \sum_{j=1, j \neq i}^N x_j^k w_{ji} \right) \quad (11)$$

where  $f$  is an activation function which is selected based on different applications.

Table 3 Weight evaluation performed by different operators

W	S	OP1	OP2	OP3	OP4	OP5	OP6	OP7	OP8	OP9	OP10	FW
$w_{1,3}$	+	S	Z	Z	Z	S	Z	S	Z	Z	S	0.100
$w_{1,4}$	+	S	M	Z	S	M	M	S	Z	M	Z	0.275
$w_{2,3}$	-	VB	B	VB	VB	VB	VB	VB	B	VB	B	-0.925
$w_{2,4}$	-	B	M	M	M	M	M	M	M	M	M	-0.525
$w_{4,3}$	+	Z	S	S	M	Z	M	Z	Z	S	Z	0.175

(S - Sign of weights; FW - Final weight values evaluated based on operator evaluation OP1-OP10 and WS)

### 3.3.4 FCMD tool design

Four nodes ( $C_1$  to  $C_4$ ) standing for the estimated contact force, the joystick moving speed change, variations of the feedback gain and its upper limit (denoted in turn as  $\delta k_f$ , and  $\delta \bar{k}_f$ ) are selected to design the FCMD tool. The values of these nodes,  $x_i$  to  $x_4$ , are then in turn the normalized values of  $\hat{F}_e$ ,  $|\dot{X}_m|$ ,  $\delta k_f$ , and  $\delta \bar{k}_f$ .

First, the influence between the FCMD nodes are analyzed. Based on the design experience, the interrelation between these nodes are defined as in Fig. 8. Due to the symmetric design of weight factors, their amplitude tagged as ‘INF’ can be represented by linguistic variables while their signs are defined by the direction of the connection edges in Fig. 8. These connection edge directions are determined based on experience of the experts. Here, five triangle membership functions tagged as ‘Z’, ‘S’, ‘M’, ‘B’, and ‘VB’, which in turn stand for zero, small, medium, big, and very big, are uniformly distributed within the closed universe [0, 1] to describe the influence (see Fig. 9). For decision making, it is realized that sigmoid and tansig (hyperbolic tangent sigmoid) functions are the two feasible choices.<sup>24</sup> In this case, the tansig function is selected as the FCMD activation function:

$$f(a) = \frac{2}{1 + e^{-\lambda a}} - 1 \quad (12)$$

where  $\lambda$  is the steepness parameter and selected as 2.

Therefore, the feedback gain and its upper limit can be updated for each working step as

$$\begin{cases} \bar{k}_f^{k+1} = \bar{k}_f^0 + (\bar{k}_f^0 - \underline{k}_f) \delta \bar{k}_f^{k+1} \\ \underline{k}_f^{k+1} = \underline{k}_f^0 \Big|_{FFGT} (1 + \delta k_f^{k+1}) \end{cases} \quad (13)$$

Second, the definition of the FCMD weights is carried out through an analysis on a series of teleoperation tests which were performed by 10 different operators. During the tests, each operator adjusted manually the FFG in order to make a proper force feedback sense to his/her hand. Through the tests, a survey on the impacts between the decisive factors has been performed by the operators. Based on their evaluation sheets (as summarized in columns OP1 to OP10 of Table 3), the final decision on FCMD weights is made by the defuzzification which is based on the centre average algorithm. By combining with the signs of the FCMD weights based on (column WS of Table 3), their final values are derived as the last column in Table 3.

## 4. Local Robust Adaptive Controller

The local robust adaptive controller (LRAC) is designed for the implementation to the slave manipulator and the master FFM (as the

SRAC and MRAC) to ensure their accurate position and pressure tracking performances, respectively. This LRAC is designed as a proportional-integral-derivative (PID) -based neural network in which the network is structured based on the well-known PID algorithm and their weights are trained online under a Lyapunov stability constrain.

The LRAC is generally built for a system with one control input,  $u$ , and  $n$  outputs (in this case,  $u$  is the driving command for the control valve 1 or 2;  $n = 1$ , means one output which is the piston displacement or the reflected pressure). The network composes of three layers: an input layer as a control error sequence, a hidden layer with three nodes, tagged as P, I, and D, and an output layer, which is the system control input. The error sequence is defined as  $e_k^{NN} = \{e_k^1, \dots, e_k^{n-1}\}$ , where  $e_k^i = y_k^{desired,i} - y_k^{actual,i}$ ;  $y_k^{desired,i}$  and  $y_k^{actual,i}$  are in turn the desired system response and the actual response. Define  $\{w_k^{Pi}, w_k^{Ii}, w_k^{Di}\}$  is a weight vector of the hidden nodes with respect to input  $i^{th}$ , and  $\{w_k^P, w_k^I, w_k^D\}$  is the weight vector of the output layer. By applying the PID algorithm, the weights of the output layer are selected as unit while the output from each hidden node is derived as

$$\begin{aligned} \text{Node P: } O_k^P &= \sum_{i=1}^{n-1} w_k^{Pi} e_k^i; \\ \text{Node I: } O_k^I &= O_{k-1}^I + \sum_{i=1}^{n-1} w_k^{Ii} e_k^i; \\ \text{Node D: } O_k^D &= \sum_{i=1}^{n-1} w_k^{Di} e_k^i + \sum_{i=1}^{n-1} w_{k-1}^{Di} e_k^i \end{aligned} \quad (14)$$

Then, the output from the network is obtained using a linear function:

$$u_k = f(O_k^{NN}) \equiv O_k^{NN} = O_k^P + O_k^I + O_k^D \quad (15)$$

To ensure the adaptability of the LRAC, the back-propagation and gradient descent method is employed to tune the network weights. Additionally, a Lyapunov stability condition is integrated with the learning algorithm to guarantee the system robustness.

Learning algorithm: define a prediction error function as

$$E_k^{NN} = 0.5 \sum_{i=1}^{n-1} (y_k^{desired,i} - y_k^{actual,i})^2 = 0.5 \sum_{i=1}^{n-1} (e_k^i)^2 \quad (16)$$

$$e_k^i \equiv y_k^{desired,i} - y_k^{actual,i}$$

The hidden weights can be online tuned for each step,  $(k+1)^{th}$ , as:

$$\begin{cases} w_{k+1}^{Pi} = w_k^{Pi} + \Delta w_k^{Pi} = w_k^{Pi} - \eta_k^{Pi} \partial E_k^{NN} / \partial w_k^{Pi} \\ w_{k+1}^{Ii} = w_k^{Ii} + \Delta w_k^{Ii} = w_k^{Ii} - \eta_k^{Ii} \partial E_k^{NN} / \partial w_k^{Ii} \\ w_{k+1}^{Di} = w_k^{Di} + \Delta w_k^{Di} = w_k^{Di} - \eta_k^{Di} \partial E_k^{NN} / \partial w_k^{Di} \end{cases} \quad (17)$$

where  $\eta_k^{Pi}$ ,  $\eta_k^{Ii}$ ,  $\eta_k^{Di}$  are the learning rates within [0,1]; the other factors in Eq. (17) are derived using partial derivative of the error function Eq. (16) with respect to each decisive parameter and chain rule method.<sup>25</sup>

**Theorem 1:** by selecting properly the learning rates in Eq. (17)  $\eta_k^{Pi} \equiv \eta_k^{Ii} \equiv \eta_k^{Di} \equiv \eta_k$  for step  $(k+1)^{th}$  to satisfy Eq. (18), then the stability



of the LRAC is guaranteed.

$$\sum_{i=1}^{n-1} (e_k^i F_k + 0.5 F_k^2 \eta_k) \leq 0 \quad (18)$$

$$\text{where } F_k = -\sum_{j=1}^{n-1} \left( \frac{e_k^j}{w_k^{Pj}} \frac{\partial E_k^{NN}}{\partial w_k^{Pj}} + \frac{e_k^j}{w_k^{Ij}} \frac{\partial E_k^{NN}}{\partial w_k^{Ij}} + \frac{e_k^j}{w_k^{Dj}} \frac{\partial E_k^{NN}}{\partial w_k^{Dj}} \right).$$

Proof: by defining a Lyapunov function as Eq. (19), the change of this function is derived as Eq. (20)

$$V_k^{NN} = 0.5 \sum_{i=1}^{n-1} (y_k^{\text{desired},i} - y_k^{\text{actual},i})^2 = 0.5 \sum_{i=1}^{n-1} (e_k^i)^2 \quad (19)$$

$$\begin{aligned} \Delta V_{k+1}^{NN} &= 0.5 \sum_{i=1}^{n-1} \left( (e_{k+1}^i)^2 - (e_k^i)^2 \right) \\ &= \sum_{i=1}^{n-1} \left( e_k^i \Delta e_k^i + 0.5 (\Delta e_k^i)^2 \right), \quad (e_{k+1}^i = e_k^i + \Delta e_k^i) \end{aligned} \quad (20)$$

From the controller structure, one has:

$$\Delta e_k^i = \sum_{j=1}^{n-1} \left( \frac{\partial e_k^i}{\partial w_k^{Pj}} \Delta w_k^{Pj} + \frac{\partial e_k^i}{\partial w_k^{Ij}} \Delta w_k^{Ij} + \frac{\partial e_k^i}{\partial w_k^{Dj}} \Delta w_k^{Dj} \right) \quad (21)$$

here, the terms  $\Delta w_k^{Pj}$ ,  $\Delta w_k^{Ij}$ , and  $\Delta w_k^{Dj}$  are obtained from Eq. (17). By using partial derivative and selecting  $\eta_k^{Pi} \equiv \eta_k^{Ii} \equiv \eta_k^{Di} = \eta_k$ , Eq. (21) becomes:

$$\Delta e_k^i = \eta_k F_k \quad (22)$$

From Eqs. (22), (20) is rewritten as

$$\Delta V_{k+1}^{NN} = \sum_{i=1}^{n-1} (e_k^i F_k + 0.5 F_k^2 \eta_k) \quad (23)$$

The tracking performance is guaranteed to be stable only if  $\Delta V_{k+1}^{NN} \leq 0, \forall k$ . It is clear that except  $\eta_k$ , the other factors in Eq. (23) can be determined online based on the prediction error and the chain rule method.<sup>25</sup> Hence for each working step, it is easy to select a proper value of  $\eta_k$  to make Eq. (18) satisfy. Therefore, the proof is completed.

## 5. Experimental Results and Discussions

### 5.1 NNEC training and validation

#### 5.1.1 Data acquisition

In order to utilize the NNEC supporting the SFFJC design, it is necessary to acquire data of the slave manipulator of the teleoperation test rig (see Fig. 3(d)) to characterize the environments as well as to train the NNEC. Open loop teleoperation experiments in which the input was the manipulator open loop driving command and the output was the manipulator displacement were performed on the test rig without activating the FFM and FSSC. Furthermore, to identify offline the environment characteristics, the pressure sensor attached to the slave cylinder was only employed for these tests to compute the interaction force between the slave and environment (Fig. 3(d)).

For each test, a trajectory was randomly given to the slave manipulator by the PC and the sampling period was set to 0.01 s. Besides the test in the free load condition (without attaching a spring), three among the five spring types in Table 1, type 1, 3 and 5, were selected to generate the three different environment classes for the

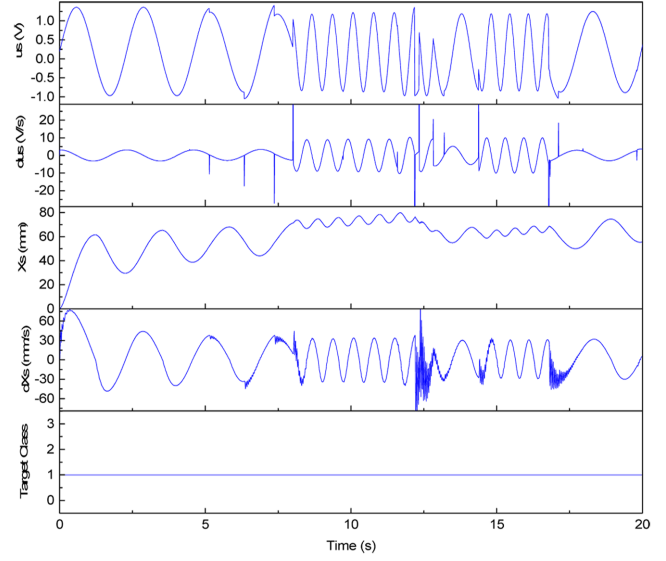


Fig. 10 Data acquisition result with respect to spring type 1

other three tests. The remained spring, types 2 and 4, were kept for the SFFJC validation to show the capability of this approach. Subsequently, the four sets of the slave input-output training data were observed. Fig. 10 displays a sample of open loop test result with respect to the spring type 1.

#### 5.1.2 Environment characterization

Next, the acquired data sets were used to characterize the corresponding environments. In this case, the recursive least square method (RLSM) was applied to identify the stiffness of the selected spring types (assumed to be unknown as the practical applications). The environment model could be represented as

$$F_e = k_e X_s + F_{e0} \quad (24)$$

where  $F_{e0}$  was the nonzero amount of the loading force at the initial position of the slave.

For each environment, by replacing  $P$  sets of input-output data points of the slave positions and loading forces observed at  $P$  sampling intervals into Eq. (24), a matrix relation was obtained:

$$AX = B \quad (25)$$

where:  $X$  was an unknown column vector including the parameters,  $k$  and  $F_0$ ;  $B$  was the loading force vector;  $A$  was a  $P \times 2$  matrix of which each row was described as  $[X_s^p \ 1]$  (the superscript  $p$  denotes  $p^{\text{th}}$  sample of the slave position).

Let define row  $i^{\text{th}}$  of matrix  $A$  in Eq. (25) as  $a_i$  and element  $i^{\text{th}}$  of vector  $B$  as  $b_i$ , by using the RLSM,<sup>26</sup>  $X$  could be calculated iteratively as:

$$\begin{aligned} X_{i+1} &= X_i + T_{i+1} a_{i+1}^T (b_{i+1} - a_{i+1} X_i) \\ T_{i+1} &= T_i - \frac{T_i a_{i+1}^T a_{i+1} T_i}{1 + a_{i+1} T_i a_{i+1}^T}, \quad i = 0, 1, \dots, P-1 \end{aligned} \quad (26)$$

where  $T_i$  was the covariance matrix.

The initial conditions to launch the algorithm Eq. (26) were  $X_0 = 0$

Table 4 Environment characterization results

Environment class (Spring type)	Actual Stiffness N/m	RLSM-based Stiffness N/m	Identification Accuracy [%]
Class 0 (Free Load)	0	-	-
Class 1 (Spring type 1)	500	509.649	98.070
Class 2 (Spring type 3)	1500	1531.047	97.930
Class 3 (Spring type 5)	2500	2550.731	97.978

Table 5 Learning success rate of NNEC [%]

Number of inputs	Number of nodes in hidden layer						
	10	15	20	25	30	35	40
20	84.43	84.66	83.71	84.67	84.82	87.12	85.71
24	86.44	84.91	87.24	93.57	94.54	90.33	85.96
28	86.25	83.82	88.04	88.35	95.49	91.57	85.77
32	85.44	84.90	88.32	90.07	92.83	91.92	86.34
36	84.15	85.88	89.82	91.58	91.32	92.83	85.41
40	84.16	85.40	82.25	84.08	83.32	83.68	83.37

Goodness of fit [%]

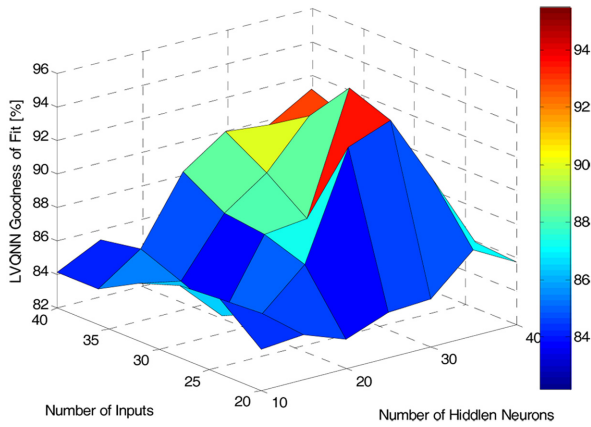
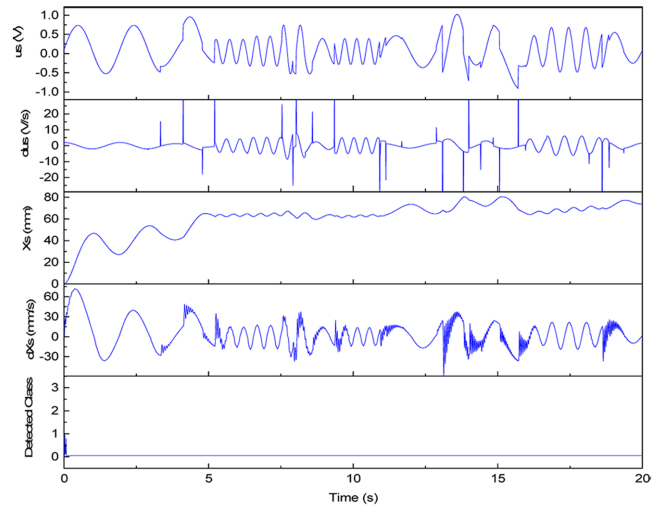


Fig. 11 Goodness of fit - 3D map

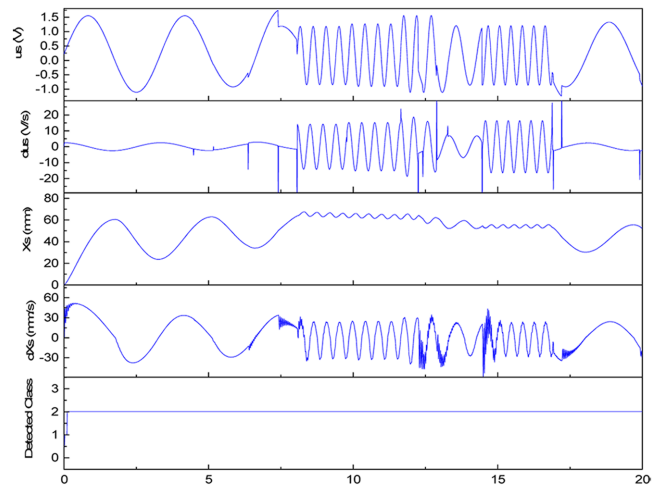
and  $T_0 = \gamma I$ , where  $\gamma$  was a positive large number and  $I$  was the identity matrix of dimension  $2 \times 2$ . Subsequently, the environment was characterized for each test case as shown in Table 4. By comparing with the actual spring stiffness, the results pointed out that the RLSM could identify precisely the environment characteristics. The results in Table 4 were then used to combine with the output from the NNEC (the detected environment class) to estimate the interaction between the slave and environment.

**5.1.3 NNEC optimization**

Next, the training progress was carried out for the NNEC by using the data sets obtained in Section 5.1.1 and the environment identification results derived in Section 5.1.2. The determination of suitable size of the input vector and number of hidden neurons in the competitive layer is one of the essential issues in practical implementation of the neural network-based classifier (Section 3.2). Here, the input vector size was defined based on the numbers of time-based data points of the available signals,  $u_s$  and  $X_s$ . The training was performed with the different settings of the input layer and hidden layer, in which the number of inputs was changed from 20 to 40 and the number of hidden neurons was varied from 10 to 40. The correlation between the network output and target output (the environment class) was used to describe the



(a)



(b)

Fig. 12 Classification result using the optimized NNEC: (a) Environment class 0, (b) Environment class 2

training success. The training result (goodness of fit [%]) of the NNEC was then analyzed in Table 5 and Fig. 11.

Table 6 Setting parameters for test profiles and PID controllers

Plant	Reference / PID gains	Multi-step reference: Step for PID Tuning	Sinusoidal reference: (amplitude - A, frequency - F)	
			$(A_1, F_1)$	$(A_2, F_2)$
Master FFM (Pressure control)	Reference	Step 1.5 bar	(2.5 bar, 1 Hz)	(2.5 bar, 2 Hz)
	$K_P$	0.3037	0.3195	0.5709
	$K_I$	0.0029	0.4305	0.4417
	$K_D$	0.0744	0.0180	0.0243
Slave Cylinder (Position control) (Environment class 1)	Reference	Step 20 mm	(20 mm, 1 Hz)	(20 mm, 2 Hz)
	$K_P$	0.1285	0.1606	0.3990
	$K_I$	0.0316	0.0240	0.0379
	$K_D$	0.0494	0.0593	0.0689

From these results, it can be found that the most suitable NNEC structure was realized with 28 nodes in the input vector and 30 nodes in the competitive layer. The learning success rate in this case was highest with 95.49 [%]. The result implies that the designed NNEC could estimate well the working environment at the slave site.

#### 5.1.4 NNEC validation

To validate the applicability of the optimized NNEC, real-time open loop experiments on the test rig were performed. During these experiments, the cylinder was randomly driven by the PC and, the NNEC was employed to detect the environment conditions. Here, the environment was set to the class 0 (free load) and class 2 (using the spring type 3). Subsequently, the NNEC detection performances were obtained as plotted in Fig. 12(a) and 12(b). These results show that the classifier could detect the loading conditions and the outputs reached stably and accurately to the true classes in a short time. This confirms convincingly that the proposed classifier could identify online correctly the environment class. By combining with the environment characterization results (as presented in Section 5.1.2), it is therefore capable of producing precisely the environment characteristics which take an important task in setting the desired reflection force.

#### 5.2 LRAC Verification

Experiments with the FFM pressure tracking control and cylinder position tracking control were done separately to investigate the capability of the local controllers, MRAC and SRAC, respectively. For each control objective, a comparative study of the designed controller and a conventional PID controller was carried out. Due to the fixed gain use, the PID control gains were manually tuned for each given trajectory by using the following steps:

**Step 1:** the control objective was approximated by a transfer function derived from its data using system identification toolbox of Matlab<sup>27</sup>;

**Step 2:** for each tracking trajectory, the PID gains were tuned for the derived system transfer function using PID tuner of Matlab/Simulink;

**Step 3:** the PID gains obtained from Step 2 were refined through control tests with the real system.

Sinusoidal and multi-step signals were used to generate tracking profiles for the tests. Based on the test rig specifications, the parameters of these profiles were properly determined. Consequently, the PID gains were derived for each test case as shown in Table 6. It is noted that in Table 6 with the position tracking, the PID gains were tuned for the case that the environment was the class 1 (using the spring type 1).

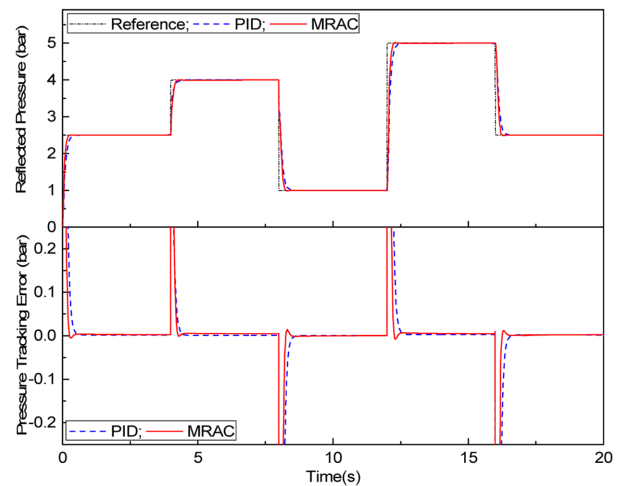


Fig. 13 Pressure tracking results: multi-step trajectory

#### 5.2.1 Rotary actuator pressure tracking control

First, experiments were carried out with the rotary actuator pressure control using the two controllers, PID and MRAC. The comparison results with respect to the multi-step and sinusoidal references were obtained and in turn plotted in Figs. 13 and 14. When dealing with the multi-step reference, the performances of both the controllers were quite similar with acceptable control error. However, by using the PID controller, of which the gains were tuned for the small jumping step (1.5 bar), the responses of the actuator to the larger jumping steps were slower than those of using the MRAC. The differences between these control performances were bigger in case of sinusoidal tracking. Although the tuned PID controller could drive the system to follow the desired goal quickly with acceptable error, this steady state error (SSE) was larger than that of the MRAC controller ( $\pm 0.2$  bar compared to  $\pm 0.05$  bar in case of the 1 Hz reference). The PID performance was continuously degraded at the higher frequency, 2 Hz with larger SSE.

Meanwhile, the MRAC could always ensure the fast response and small SSE in both the test cases (less than  $\pm 0.075$  bar, equivalent to  $\pm 3\%$  of the sinusoidal trajectory amplitude). The reason was that the MRAC was the advanced combination between the adaptive neural network and robust learning technique to compensate the system nonlinearities and uncertainties and therefore, minimize the control error.

#### 5.2.2 Cylinder position tracking control

Next, experiments were carried out with the cylinder position control

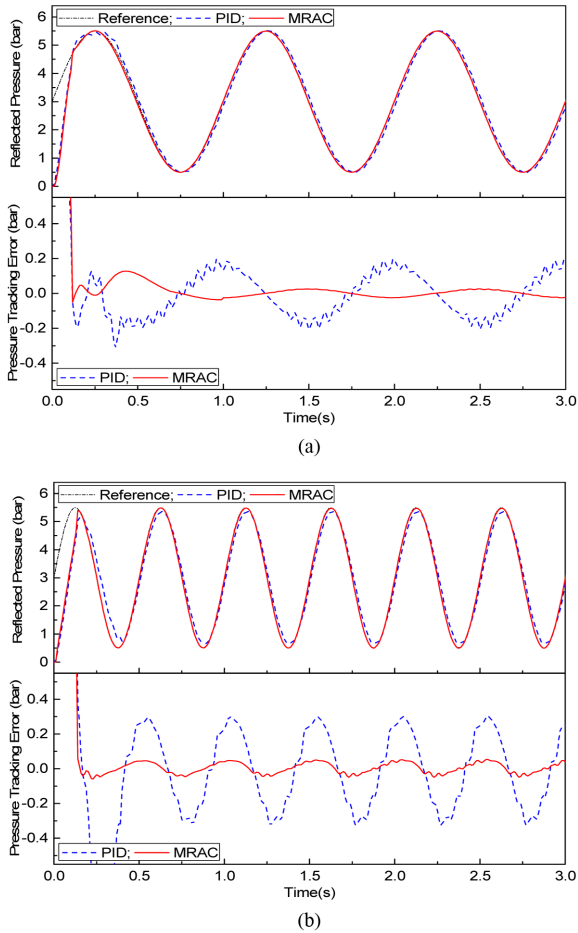


Fig. 14 Pressure tracking results: (a) Sine 1 Hz, (b) Sine 2 Hz

using the two controllers, SRAC and PID (Table 6). Similar to the pressure tracking control tests, the two controllers were in turn applied to drive the cylinder to follow the trajectories defined in Table 6 in different load conditions. Herein, the environment classes 1 and 3 in Table 4 were employed for the tracking control tests. The comparison results were obtained as plotted in Figs. 15 to 17. As denoted in Table 6, the PID control gains were derived for the system working in the environment class 1. Hence, when the environment class 1 was selected, the tracking results using the compared controllers were both acceptable as shown in Fig. 15. The PID controller, with the gains optimized manually for the working step 20 mm and environment class 1, could achieved the small SSE in the small working steps. This SSE was increased together with the rise of undershoots when the system faced with the large step changes. Similarly, the control performance was degraded when the external environment was varied from the class 1 to class 3. This phenomenon is clearly shown through the PID control results with respect to the sinusoidal references.

Without changing the environment condition, compared to the condition to derive the PID gains, the controller could drive the system to reach the goals with small SSE (within  $\pm 1.2$  mm, corresponding to 6% of the reference amplitude) in both the two working frequencies (Figs. 16 and 17). Once the environment was changed, the performance was significantly deteriorated with large SEE (around  $\pm 3.8$  mm, corresponding to  $\pm 19\%$  of the reference amplitude). On the contrary,

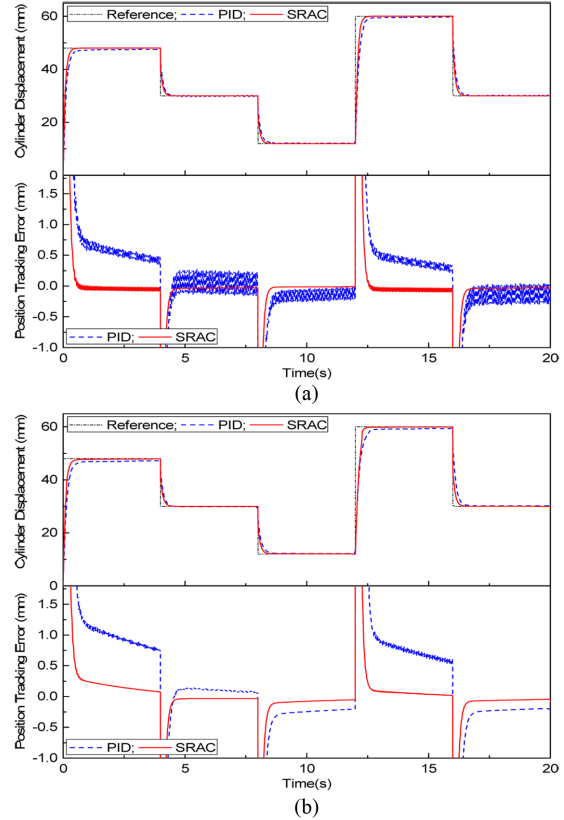


Fig. 15 Position tracking results: multi-step trajectory (a) Environment class 1; (b) Environment class 3

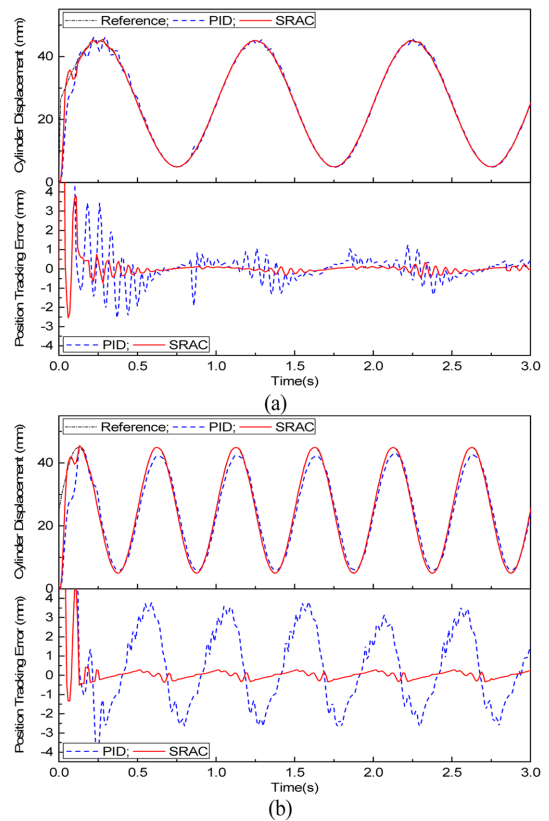


Fig. 16 Position tracking results - Environment class 1: (a) Sine 1 Hz, (b) Sine 2 Hz

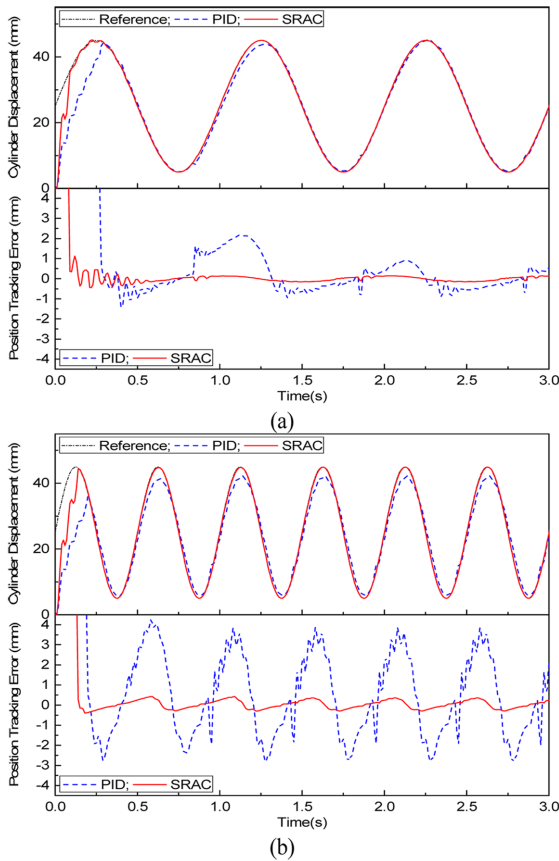


Fig. 17 Position tracking results - Environment class 3: (a) Sine 1 Hz frequency, (b) Sine 2 Hz

by using the SRAC with the same advanced functionalities as the MRAC, the best tracking results with small overshoot, fast rising time and small SSE were achieved in both the cases (Figs. 15 to 17). The tracking accuracy using the SRAC was always guaranteed to be less than  $\pm 0.5$  mm ( $\pm 2.5\%$  of the trajectories' amplitudes).

From the comparative studies with both the local controllers, SRAC and MRAC, it can be concluded that these designed controllers are powerful for the teleoperation control application.

### 5.3 SFFJC verification

In this section, the full control system, SFFJC, has been applied to the test rig (Fig. 3) in order to evaluate its applicability to teleoperation applications. A series of experiments on the testing system was conducted under the three different environment classes. Here, to investigate the adaptability of the SFFJC, the spring types 1, 2, and 4 from Table 1 were chosen to represent the environment. Additionally, in order to create challenges for the control system in detecting the environment changes as well as keeping the stability of the FFM rotary actuator and slave manipulator, these springs were installed so that they were initially in the free lengths (by adjusting the lock position of the slider in Fig. 3(d)). The springs were only compressed when the cylinder rod moves forward a pre-defined distance. In this study, this distance was set to 7 mm. The joystick commands were randomly given by the operator to drive the slave manipulator. Based on an analysis of the operator-joystick and slave dynamics and using the trial-and-error method, the

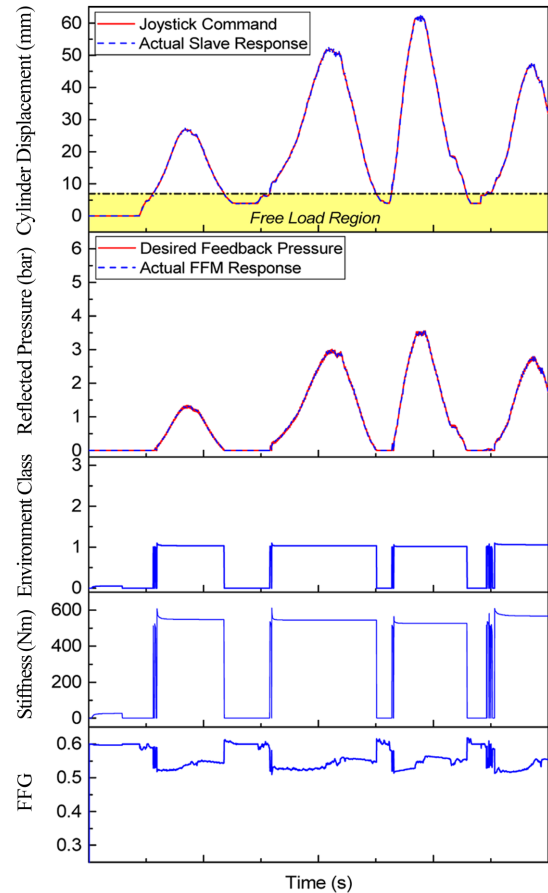


Fig. 18 Teleoperation performance in environment class 1

transformed factor was properly assigned by  $1/4.5$ , which means a 4.5 N reflected force was equivalent with a 1 bar reflected pressure.

The closed loop teleoperation experiments were then performed and the results are plotted in Figs. 18 to 20. From these figures, it can be seen that the proposed SFFJC behaved well with high accuracy. This comes as no surprise because the SFFJC possesses the three advanced modules: FSSC, MRAC and SRAC. With the FSSC implementation, the environment characteristics in either free-load or load condition could be estimated accurately by the NNEC. As shown in the third and second sub-plots from the bottom of Figs. 18-20, the environment classes were determined correctly as 1, 1.5 and 2.5 which were correspond with the actual spring selection, type 1, type 2 and type 4. Moreover, the NNEC could detect immediately the change of external environment, from free load (indicated by the free load region in the top sub-plots) to load and vice versa. Based on this estimation, the FFFC was capable of producing properly the desired reflected pressure by regulating smoothly the FFG in order to make the similar load feeling to the master site (see the first sub-plots from the bottom of Figs. 18-20). Next, the two local controllers took parts in executing precisely the tasks given to the slave manipulator and the master FFM (cylinder position control and rotary actuator pressure control). The control results, which are in turn depicted in the fourth and fifth sub-plots of Figs. 18-20 (from the bottom), proved remarkably the capability of these controllers. As a result, the proposed SFFJC could ensure the stable performance for the teleoperation system in which the slave



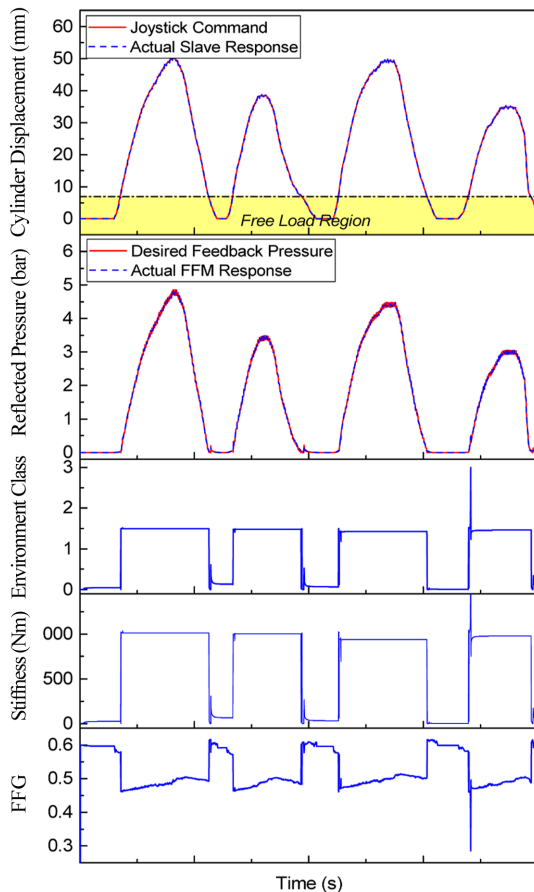


Fig. 19 Teleoperation performance in environment class 1.5

performed accurately the desired task while the operator was able to realize truthfully the interaction between the slave manipulator and the environment.

## 6. Conclusions

This paper presents the simple, safe and cost-effective approach named sensorless force feedback joystick control for teleoperation applications, especially in construction sector. The main contributions of this study can be summarized as follows:

- + The force sensorless supervisory controller is designed as the combination of the neural network-based environment classifier and the fuzzy-based force feedback tuner. The NNEC is capable of detecting the environment characteristics at the slave site without requiring any sensor to estimate the interaction between the slave and environment. Meanwhile, the FFFT with the fuzzy-based cognitive map decision tool is capable of deriving appropriately the target reflecting pressure for the FFM based on the NNEC outputs.

- + The two robust adaptive controllers, SRAC and MRAC, are implemented to ensure the adaptability and stability of the position and pressure tracking of the closed loop slave and master systems.

- + By integrating both the advanced characteristics of the FSSC, SRAC and MRAC into the SFFJC, the acceptable teleoperation performance could be always achieved disregarding the system

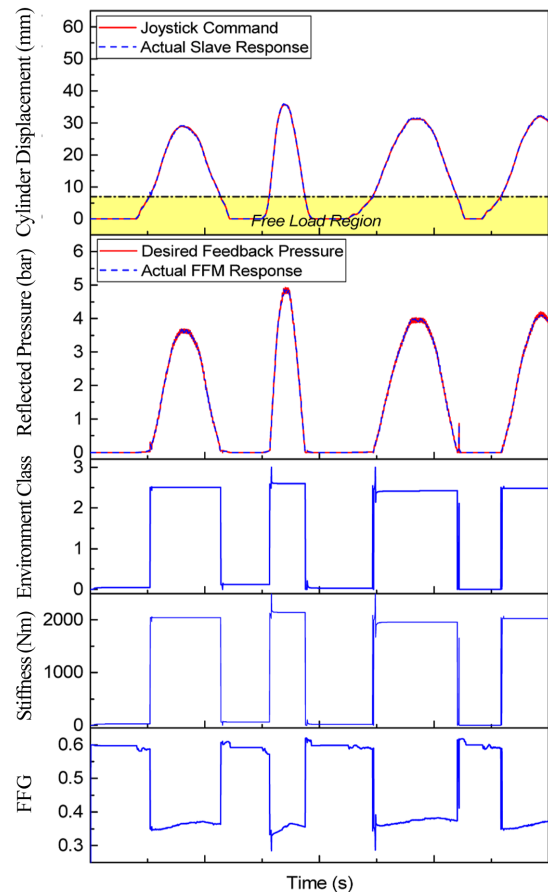


Fig. 20 Teleoperation performance in environment class 2.5

nonlinearities and uncertainties. The experimental results then prove convincingly the effectiveness of the proposed control approach.

It should be noted that the focus of this paper is to develop the SFFJC approach over wired communication. Therefore, the presence of time delays over communication channels of a teleoperation system is not considered here. The delay problems and/or data packet losses normally existing in wireless or network-based control systems have been carefully addressed and separately resolved by the authors as presented in our recent publications.<sup>28-30</sup> As an area of the future work, the development of a SFFJC-based teleoperation control system with imperfect communication using the results of our current studies is carrying out in order to widen its applicability. Another aspect of the future work is to investigate the capability of the proposed control approach when dealing with multi-DOF teleoperation systems.

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